In the Specification

On page 6, line 18 through page 13, line 28, kindly rewrite the paragraph as follows:

The drum (9) arranged at the rear is internally hollow and receives a cutting blade (13) produced in one or two parts and articulated on a blade carrier arm (14). This cutting blade is characterized by a twisted shape of the whole of its teeth, making I possible to have an effect of progressive penetration of the blade into the material from the ends toward the central zone of the latter. Each tooth is advantageously located in a plane perpendicular to the longitudinal axis of the drum. More particularly, as illustrated in the drawings, the cutting blade is designed in two parts (13.1, 13.2) fastened to the blade carrier arm (14) by means of screws of otherwise. The two blade parts meet toward the central part of the drum ~~(9.1)~~(9), while at the same time leaving a space (e) corresponding, in practice, to a cradle (15) for supporting the blade carrier arm with respect to the inner bottom of the drum. This cradle has an indentation allowing the arm to rotate on itself. At the termination of the arm, one end (14.1) of the latter has a projecting abutment (16) which comes into contact with stroke-limiting bearing planes (17) formed on the transverse face (9.2) of the drum. The other end (14.3) of the arm receives a notched collar or pinion (18) adjacent to the first toothed ring (11), as recalled above. The notched collar is mounted rotatably together with the arm. The function of this notched collar will be specified later.

The second guide drum (10) has, in its central part, a radial cavity ~~(10.1)~~(10.2) which is in the prolongation of the cradle (15) for supporting the blade carrier arm (14) of the first drum (9). This second drum (10) receives at the end a second fixed toothed ring (12) cooperating with the first fixed ring (11) in the first drum. ~~Said~~The second drum has a spindle (10.1) capable of coming into engagement in a bearing-forming recess made in the lower part of the opposite flange. ~~Said~~The spindle is capable, moreover, of receiving, between the second toothed ring (12) and the flange, a third toothed ring (19) of small diameter which is capable of cooperating with a complementary fourth toothed ring (20) mounted on the opposite flange (4). This fourth toothed ring (20) is of larger diameter in a much higher dimensional ration, and comprising a number of teeth corresponding to the dimension of the format of the strip of material to be cut. ~~Said~~The

fourth toothed ring (20) is capable of cooperating with a no-return pawl (22) mounted on a pin (23) fastened to the flange (4) together with an abutment (22.1). The fourth toothed ring (20) receives, on its face (20.1) opposite the flange, a profiled fixed cam (24), the function of which will be described later. The fourth ring (20) has, on its face (20.2) toward the drums (9 – 10), an orifice (20.3) allowing the positioning of a finger (342.1) of a link (42) in the other end (42.2) which makes it possible to fasten a return spring (26), the other end of which is integral with a fixed point ~~(27)~~(25) formed on the flange (4). The link thus ensures an eccentric function during the rotation of the ring (20) counter to the return spring (26), at the same time forming the mechanism for starting the rotation of the drums (9 – 10).

According to another ~~arrangement~~feature, the shaft (9.1) of the drum (9) is prolonged on the flange side so as to receive a movable cam (27) articulated on the shaft, this cam being profiled in a specific way in order to cooperate, in certain operating phases, with the abovementioned first cam (24).

According to another ~~arrangement~~ feature of the invention, the two drums (9 – 10) have, near their respective first and second toothed rings ~~(10 – 11)~~(11 – 12), two recesses (47), opposite or not, allowing the temporary insertion of a template-forming caliper (29), the end of which is designed in the form of a fork so as to penetrate into the recesses (47). The function of this caliper is to make it possible to set the various toothed rings in position in relation to one another as a function of the format of the strip of material to be dispensed. This template has a spindle (29.1) capable of engaging into the central axial orifice (20.4) of the ring (20). ~~Said~~This template also has an index (29.2) engaging into an orifice (20.5) formed on the ring (20). This orifice is arranged in a special way for format setting. It defines the position of the toothed ring (20) supporting the fixed cam (24), this being in relation to another movable cam (27) which will also be described.

According to another ~~arrangement~~feature, the drum (10) is capable of receiving in its radial cavity ~~(10.1)~~(10.2), with free and controlled articulation, a fork-shaped attached tab (30) integral with ~~a~~an upper flap (31), this flap being capable, after the positioning of the tab by

latching on the drum (10), of covering the drum (9) completely and the drum (10) partially. ~~Said~~ This flap has on the inside uniformly arranged projecting ribs (31.1), the function of which is to ensure the guidance of the strip of material during cutting.

According to another ~~arrangement~~feature, a lower flap (32) is mounted so as to be articulated from a front spacer bar (43) formed between the flanges (4 – 5). This lower flap has a curved shape capable of substantially surrounding the drum (10) over an angular sector and from underneath the drum (10). This flap opens at the front of the appliance and allows the introduction and guidance of the free end of the strip of material coming from the reel.

According to another ~~arrangement~~feature, the drum (9) has, at the termination of its shaft, on the opposite side to said toothed rings, a bearing surface (9.5) capable of receiving fixedly a large-diameter loading wheel (33) projecting externally from the cartridge and from the appliance, in order to allow manual control for the user to carry out the loading or fault correction of the reel of material to be dispensed.

Referring to the drawings, in particular figure 2, the travel of the strip of material has been illustrated.

The reel of material can be put in place between the supporting flanges in one direction or the other, that is to say the strip unwound from the reel is located either at the front of the appliance, as illustrated in figure 2, or toward the rear of the latter (not illustrated). Said strip is then introduced between the roller (10) and the retractable lower flap ~~(31)~~(32), at the same time passing in front of the appliance, and then the strip is introduced and guided so as to pass between the two drums (9 – 10) in order to be would partially onto the drum (9) comprising the cutting device. A guide (38) integral with the front space bar (430 ensures the upward return of the strip of material between the drums (9 – 10). The strip of material emerges at the rear of the appliance, at the same time being guided by the articulated flap (31).

According to another ~~arrangement~~feature, the blade carrier arm (14) receives, at the termination of its end, athe pinion (18) which is capable of cooperating with a complementary pinion (36) formed laterally on the movable cam (27). Thus, the displacement and emergence of the cutting blade give rise, in parallel and simultaneously, to the positioning of the movable cam under the conditions which will be explained below with reference to drawings 7, 8, 9 and 10.

The movable cam (27) has a specific configuration, with a semicircular rear part (27.a) prolonged by a first rectilinear slope (27.b) and a second rectilinear slop (27.c), but oriented angularly with respect to the first slope, so as to approximate a beak shape (27.d) continued by a connecting line (27.e) to the rear part of the cam.

The fixed cam (24) arranged eccentrically on the fourth inner ring is of smaller dimension and is bean-shaped, in particular with a rectilinear base (24.1).

It is appropriate, at this juncture, to describe the functioning of the appliance.

The loading of the reel involves introducing the free end of the latter between the retractable lower flap (32) and the drum (10) which, for this purpose, advantageously has a gripping surface. The strip is guided between the two drums (9 – 10), and then, by virtue of the bearing effect of the upper flap (31), said strip is wound partially around the drum (9) so as to emerge behind the latter. In the initial state, the cutting blade is integrated in its drum (9), without emerging. The return spring is not subjected to extension stress. When the strip of material is pulled by the user, this gives rise progressively and simultaneously to the rotation of the drum (9) which meshes with the drum (10) by means of respective first and second rings ~~(10 —11)~~(11 – 12). The third ring (19), adjacent to the ~~first~~second ring ~~(11)~~(12), causes the rotation of the fourth ring (20). The no-return pawl, penetrating into the teeth of the ring (20), locks the mechanism in position, while preventing a return into position. The fourth ring (20) rotates. The connecting and fastening point of the link (42) moves away, at the same time causing the extension of the return means (26) until the latter exceeds a dead center corresponding to the maximum distance of the fastening point of the link with respect to the fixed fastening point

formed on the flange. The rotation of the toothed ring ~~(10)~~(11) simultaneously brings about the rotation of the notched collar (18) mounted on the blade carrier arm and therefore the progressive emergence of the cutting blade from the drum (9). In practice, the emergence of the cutting blade takes place at the termination of the second revolution of the drum (9) on itself, in particular with regard to a format of 25 centimeters. At maximum dead center, the cutting blade has fully emerged, and the exceeding of dead center causes the return of the mechanism and of the cutting blade into the initial position, along with a corresponding cutting of the strip of material.

The retraction of the cutting blade takes place as a result of the contact of the two teeth (13.3) brought closer together near the middle zone of the drum. These two teeth come into perpendicular contact with two abutments (41) which are made from rubber which, by virtue of a counterbearing effect, allow the blade carrier to tilt rearward on itself, along with the pivoting of the blades and therefore of the teeth, until they return into the drum (9).

Referring now to the two fixed (24) and movable (27) cams mentioned above, figures 7, 8, 9 and 10 illustrate the various respective positions of these. In the initial situation, figure 7, the fixed cam is away from the movable cam, and is located substantially opposite the latter. The pull on the strip of material causes the tilting of the movable cam until the position in figure 8 is reached. In this situation, the cutting blade remains retracted in the drum. After the use continues with the manual pull, the two cams come into contact with one another, figure 9, and the movable cam temporarily becomes fixed. The outer curved part of the fixed cam comes into contact with the second rectilinear zone of the movable cam, said second rectilinear zone being adjacent to the beak. The continuation of the pulling movement gives rise progressively to the escape of the movable cam until their two beaks come opposite one another and correspond to the maximum emergence of the cutting blade (figure 10). The cams resume their initial position after dead center is passed in this way.

It is now appropriate to explain another particular ~~arrangement~~ feature of the invention which makes it possible to vary the format of the strip of material to be cut.

A first possibility involves arranging on the drum (10) a toothed ring (12) defined with a suitable number of teeth for the selected format, for example 25 centimeters. This corresponds to the embodiment illustrated in figure 1. The change of format makes it necessary to change the ring (12) for another ring having a different number of teeth, and a different dimension ratio will be obtained between the rings (20 and 12), thus making it possible to obtain a different format, for example 37 centimeters. If the ring (12) is molded together with the drum (10), it is only this component which will have to be changed, the others remaining the same. It is expedient to note that, as a function of the diameter of the ring (20), two or three turns of the drum (9) will be necessary before the emergence of the cutting blade.

In an alternative embodiment corresponding to a second possibility, figures 11 and 12 illustrate an arrangement of the appliance which allows format selection directly, without the drum (10) having to be changed.

For this purpose, the drum (10) is arranged differently so as to receive a movable spindle (10.1) capable of receiving a sliding pinion (450. The movable spindle (10.1) slides axially with respect to the drum (10), while being held at its two ends, on the one hand, by the drum itself and, on the other, by an orifice formed in the abovementioned flange (4). ~~Said~~ This drum is arranged, in its end part, with a profiled inner cavity making it possible, on the one hand, to accommodate and guide the end of the spindle (10.1) and, on the other hand, to accommodate the ring (19) in a specific position. ~~Said~~ The ring (19) is integral with the movable spindle (10.1), but within a space (1.e) for moving away with respect to the sliding pinion (45).